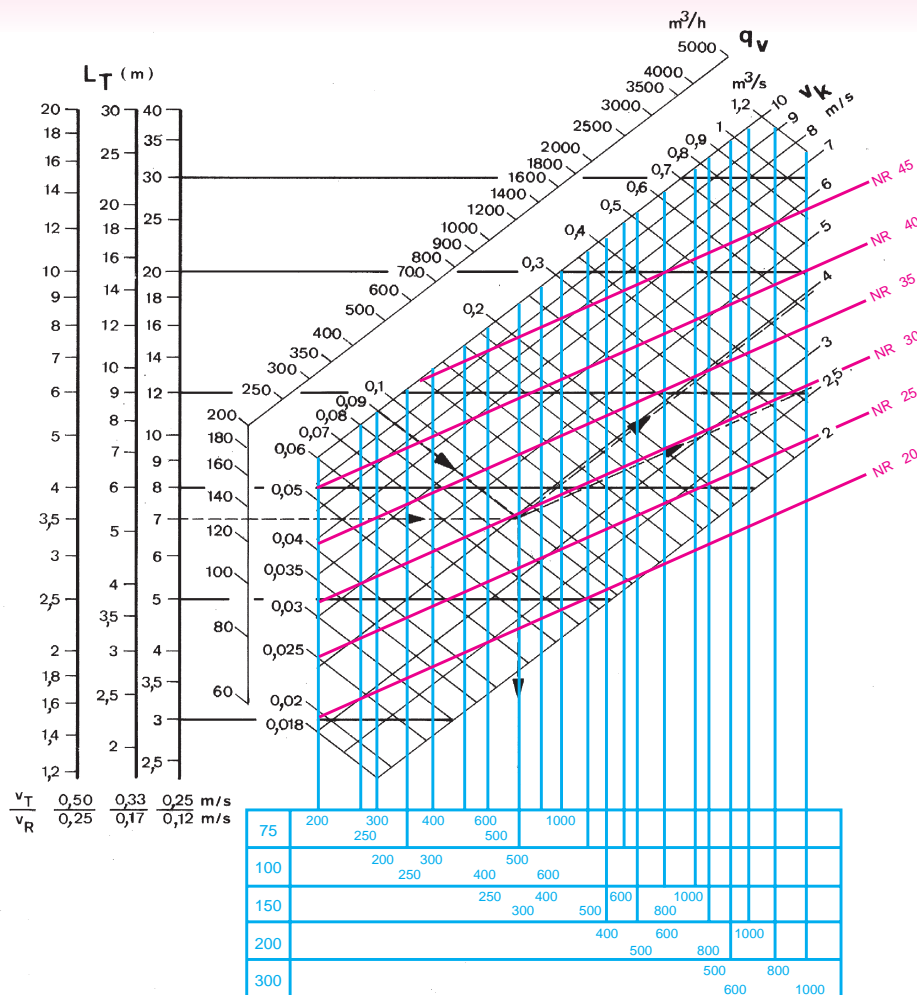


# BAR GRILLE A-300/400/500

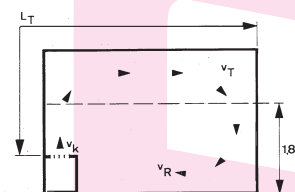
## Selection diagram - supply

0° deflection

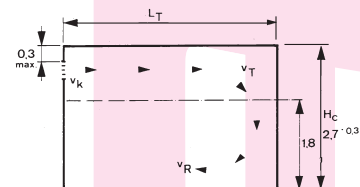
- with ceiling effect
- damper completely open



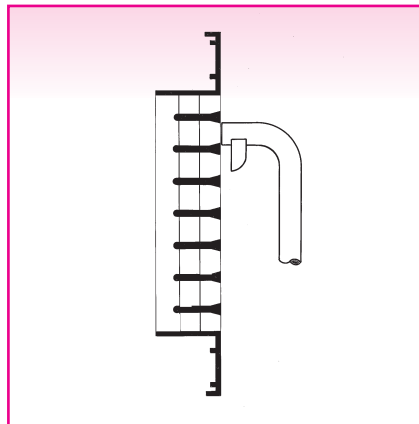
Sill mounted



Wall mounted



## Air flow rate measurement- supply



Velometer jet 2220 A or 6070

A<sub>K</sub>-values (m²)

H (mm)	L (mm)							
	200	250	300	400	500	600	800	1000
75	0,006	0,008	0,009	0,013	0,016	0,019	0,027	0,031
100	0,009	0,011	0,013	0,019	0,023	0,027	0,038	0,047
150	—	0,019	0,023	0,031	0,038	0,047	0,063	0,078
200	—	—	—	0,042	0,053	0,063	0,084	0,108
300	—	—	—	—	0,084	0,099	0,133	0,167

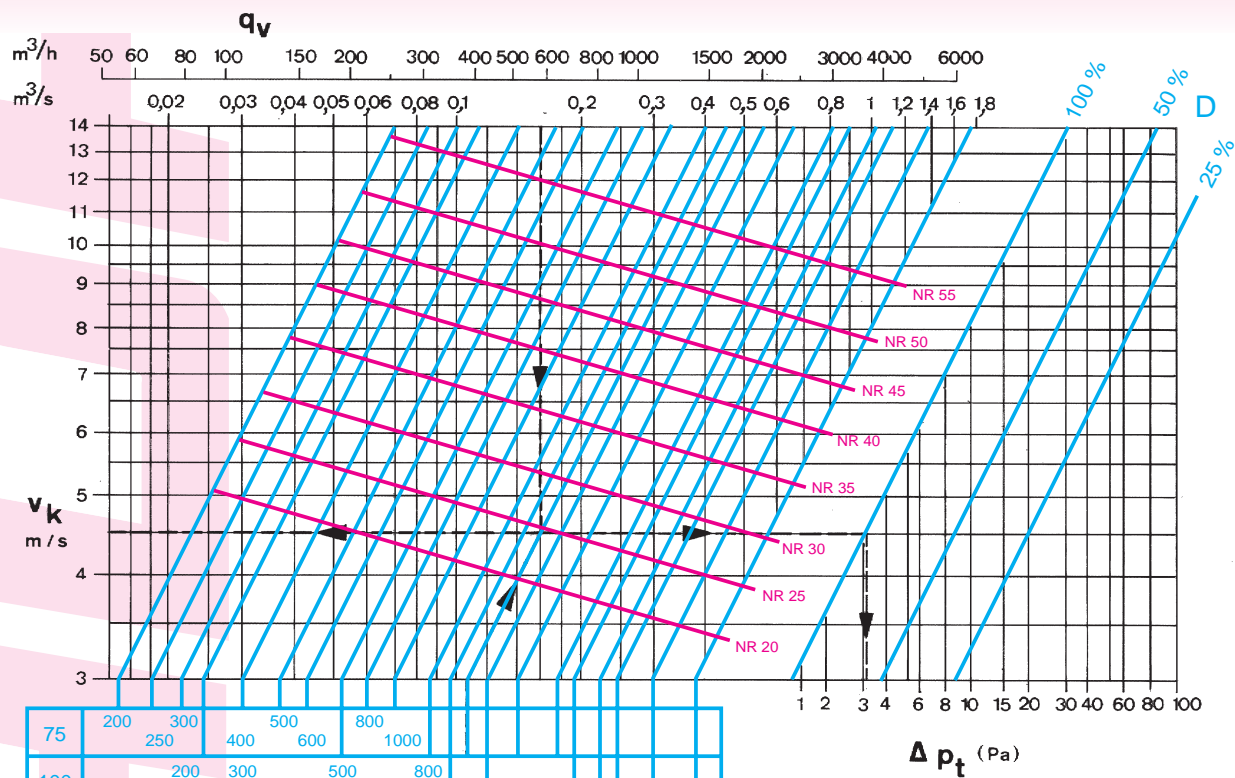
### Correction factors:

- Throw correction factor without ceiling effect

Distance between ceiling and supply grille	Correction
≥ 0,9 m	L <sub>T</sub> X 0,75

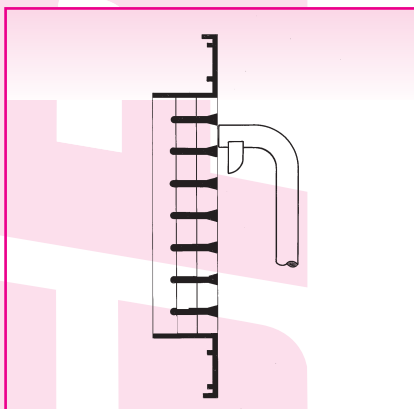
- Correction factors for vertical vane deflection of flow equalizer (see p. 1 231 verso)

## Selection diagram - exhaust



When 15° deflected bars are used, air flow rate will be reduced by 5% at listed  $\Delta p_t$  and NR values.

## Air flow rate measurement - exhaust

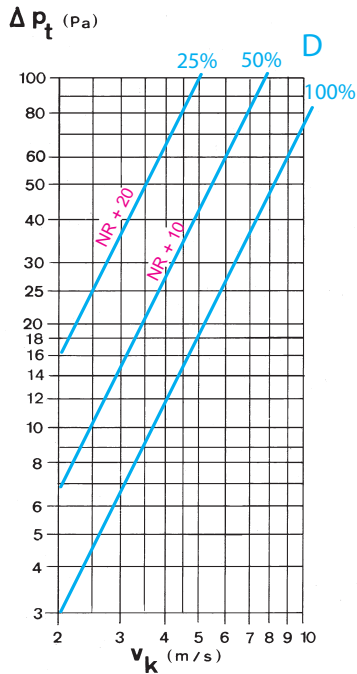


Velometer jet 2220 A or 6070

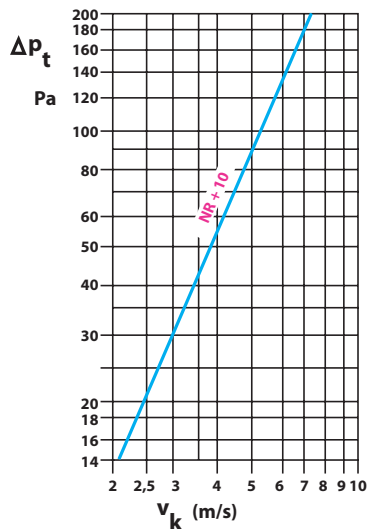
A <sub>k</sub> -values (m <sup>2</sup> )								
H (mm)	L (mm)							
	200	250	300	400	500	600	800	1000
75	0,005	0,006	0,007	0,010	0,012	0,014	0,020	0,023
100	0,007	0,008	0,008	0,014	0,017	0,020	0,028	0,035
150	—	0,014	0,017	0,023	0,028	0,035	0,047	0,058
200	—	—	—	0,031	0,039	0,047	0,063	0,080
300	—	—	—	—	0,063	0,074	0,099	0,125

## Pressure loss - supply

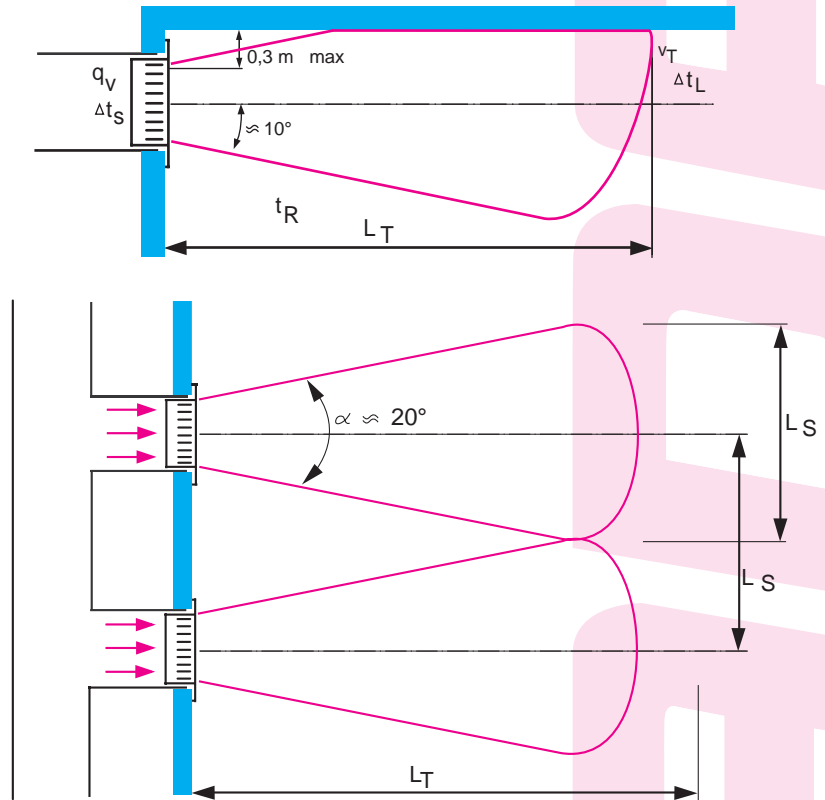
with damper type .. 7



with perforated sheet type ... 3



## Example



### SUPPLY

#### Selection data:

Air flow rate  $q_v = 0,09 \text{ m}^3/\text{s}$   
Throw  $L_T = 7 \text{ m}$  at  $v_T = 0,25 \text{ m/s}$ .

#### Solution:

Grille  $500 \times 100$  or  $300 \times 150 \text{ mm}$ .  
Supply air velocity  $v_k = 3,9 \text{ m/s}$ .  
Noise level NR 29  
Total pressure loss with perforated sheet:  $\Delta p_t = 59 \text{ Pa}$ .  
Noise level correction NR  
 $29 + 10 = \text{NR } 39$

### EXHAUST

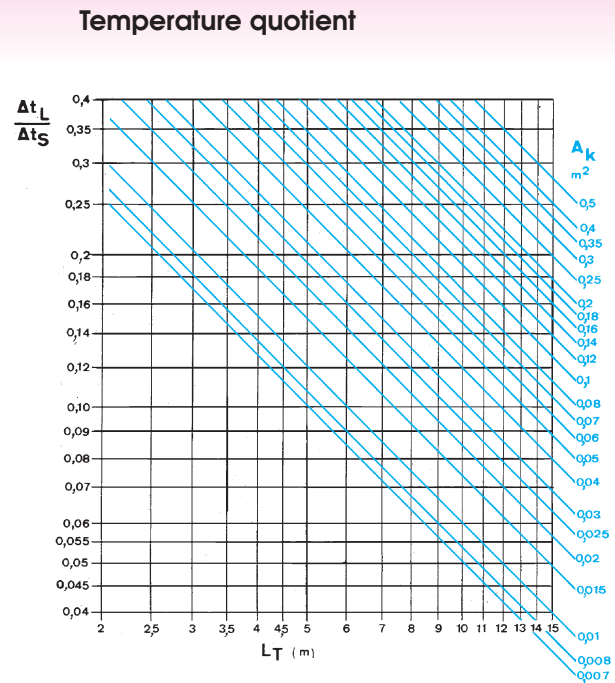
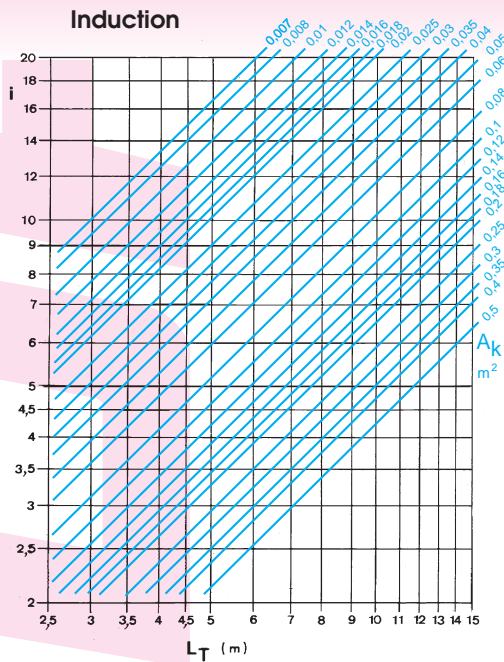
#### Selection data:

Exhaust air flow rate  $q_v = 0,16 \text{ m}^3/\text{s}$

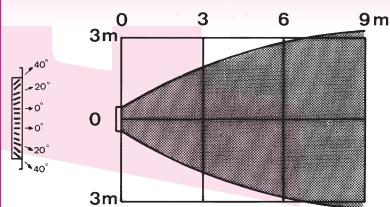
#### Solution:

Grille  $1000 \times 100 \text{ mm}$ .  
Air velocity  $v_k = 3,9 \text{ m/s}$ .  
Noise level NR 25  
Total pressure loss with damper  
100 % open:  $\Delta p_t = 3,2 \text{ Pa}$

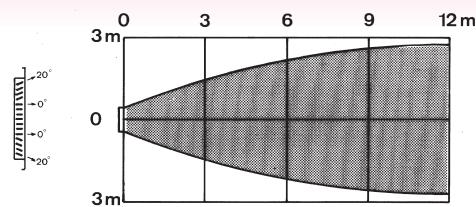
## Induction and temperature quotient with ceiling effect (also valid for linear grilles)



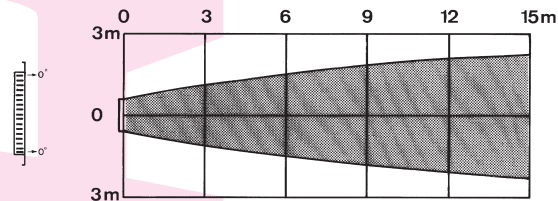
**40° deflection**



**20° deflection**



**0° deflection**

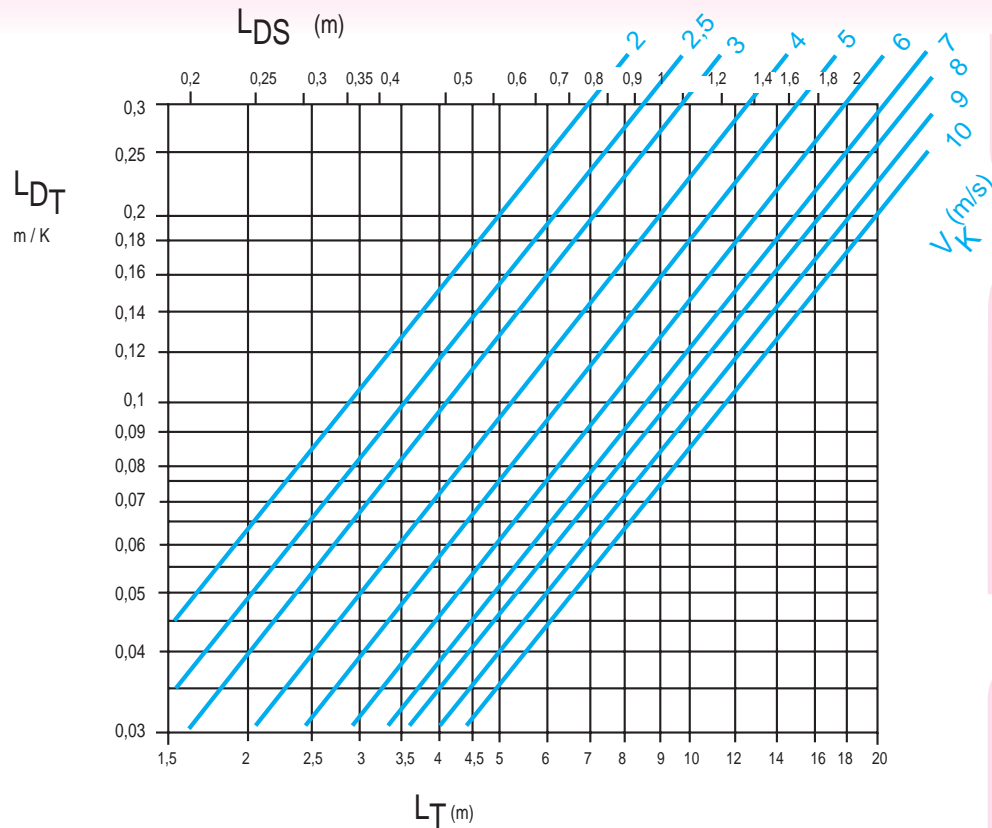


### Correction factors

Correction factors for vertical vane deflection of flow equalizer

Type	Deflection	$A_k$	$v_k$	$L_T$	NR	$i$	$\frac{\Delta t_L}{\Delta t_S}$
300, 400	20°	x 0,87	x 1,15	x 0,85	+ 3	x 1,4	x 0,7
	40°	x 0,80	x 1,25	x 0,75	+ 5	x 2	x 0,5

## Drop requirements

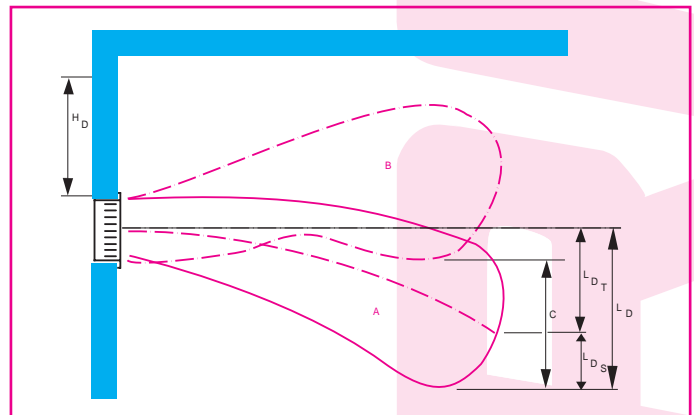


### Drop requirements

The total drop is the maximum vertical distance between the centre of a grille core and the lower point of a specified envelope, determined by the envelope velocity  $v_T$ .

The total drop consists of two elements:  $L_D = L_{DS} + L_{DT}$

- 1) The isothermal drop  $L_{DS}$  is the distance between the centre of an air current and the lowest point of the envelope.
- 2) The non-isothermal drop  $L_{DT}$  is the distance between the centre of the grille core and the air current centre line, at the place of measurement.

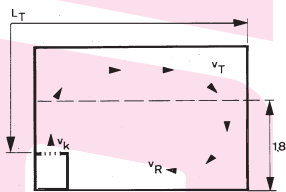


## Selection diagram - supply

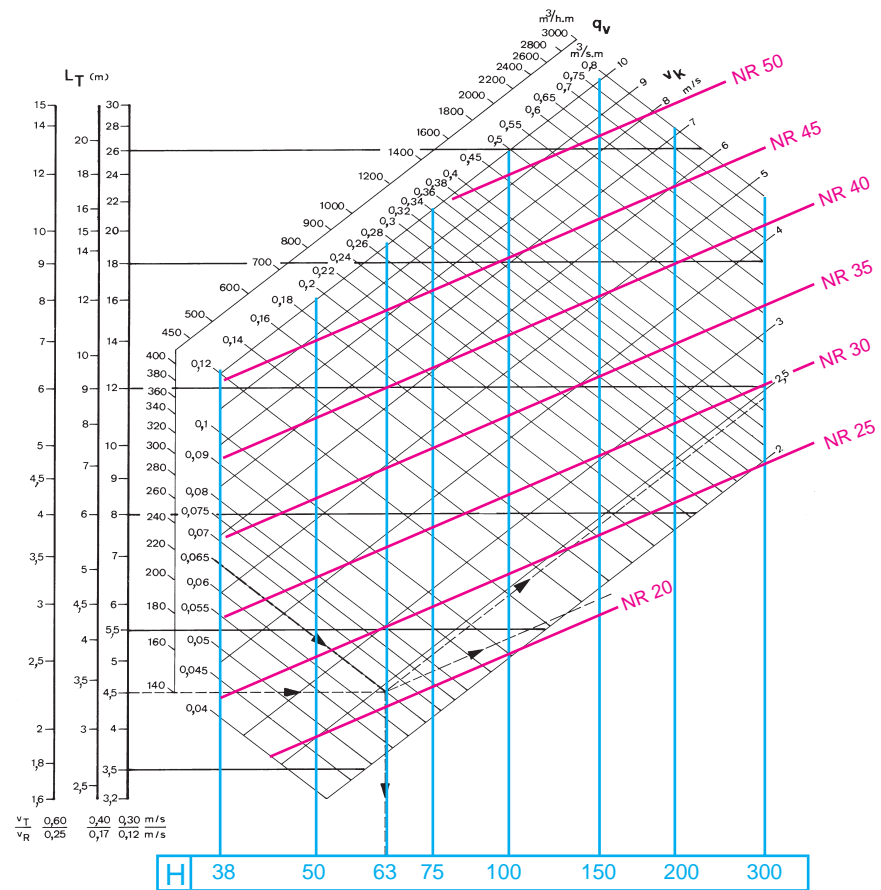
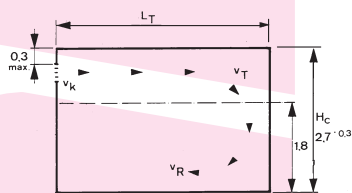
### 0° deflection

- with ceiling effect
- damper completely open

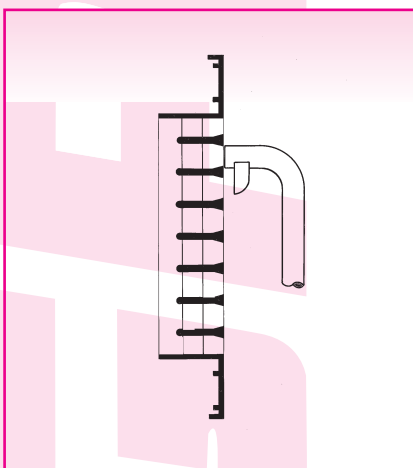
#### Sill mounted



#### Wall mounted



## Air flow rate measurement - supply

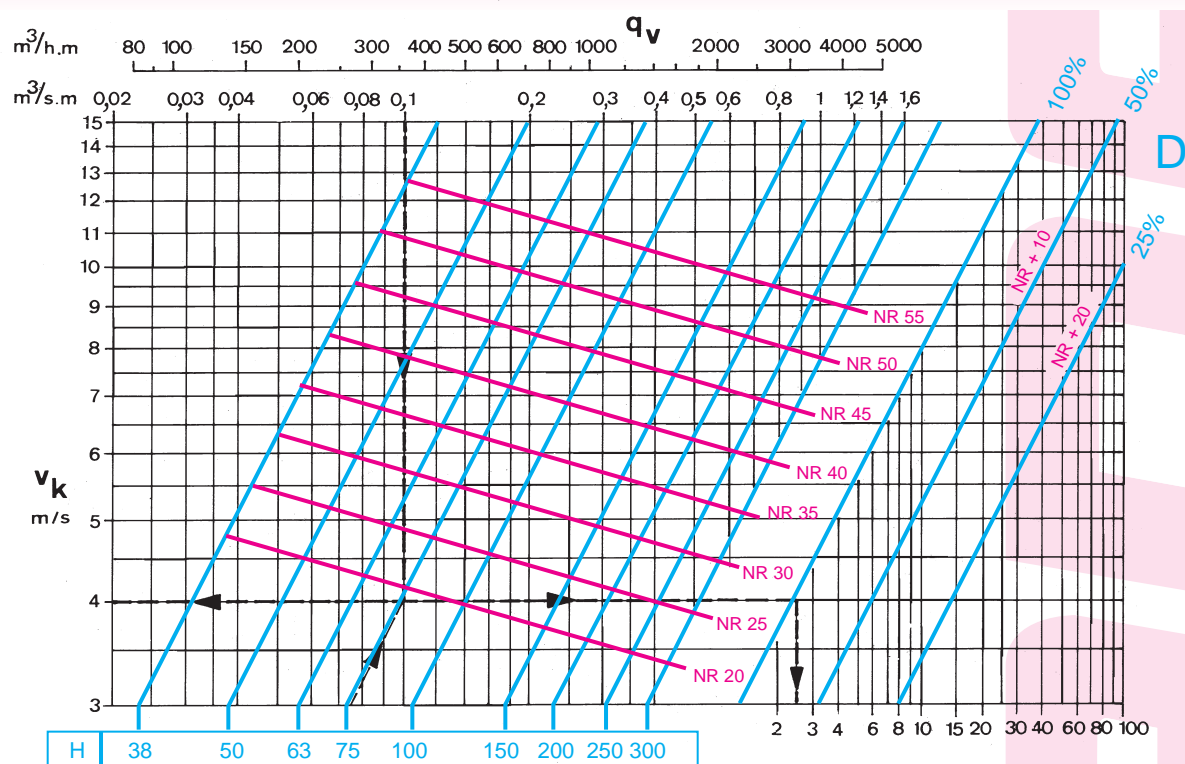


Velometer jet 2220 A or 6070

A <sub>k</sub> -values (m²/m)			
H (mm)	A <sub>k</sub>	H (mm)	A <sub>k</sub>
38*	0,012	100	0,049
50*	0,019	150	0,079
63*	0,027	200	0,110
75	0,034	300	0,171

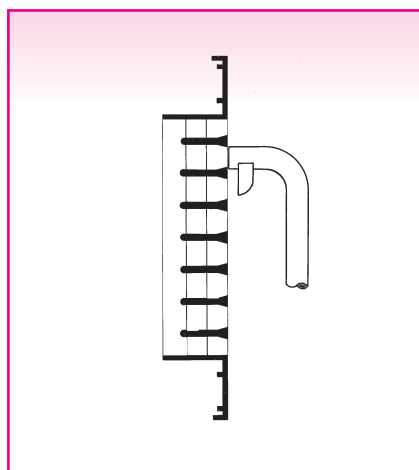
\* A-400/500 only

## Selection diagram - exhaust



When 15° deflected bars are used, air flow rate will be reduced by 5 % at listed  $\Delta p_f$  and NR values.

## Air flow rate measurement-exhaust



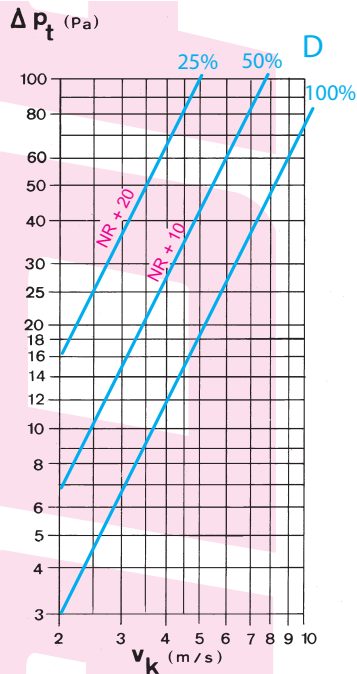
Velometer jet 2220 A or 6070

A <sub>k</sub> -values (m <sup>2</sup> /m)			
H (mm)	A <sub>k</sub>	H (mm)	A <sub>k</sub>
38*	0,008	125	0,048
50*	0,013	150	0,059
63*	0,019	200	0,082
75	0,025	250	0,105
100	0,036	300	0,127

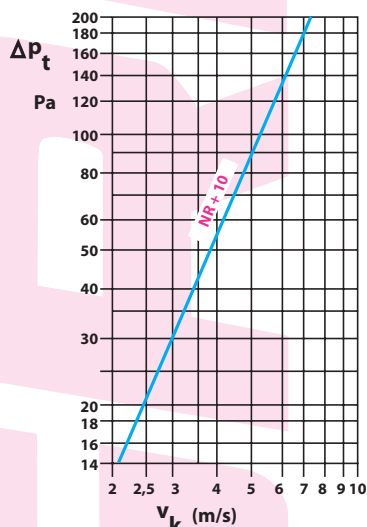
\* A-400/500 only

## Pressure loss - supply

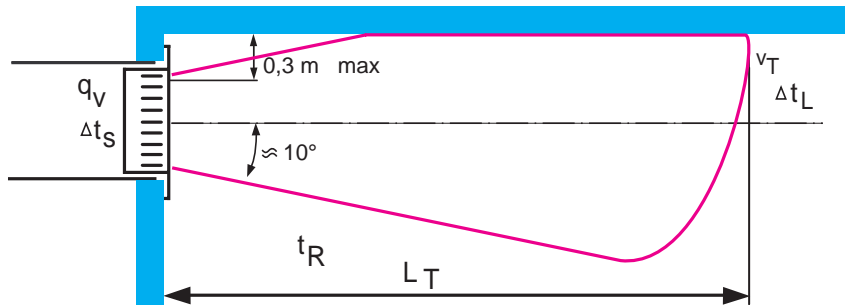
with damper type .. 7



with perforated sheet type .. 3



## Example



### Correction factors - supply:

- Throw correction factor without ceiling effect

Distance between ceiling and supply grilles	Correction
$\geq 0,9$ m	$L_T \times 0,75$

- Correction factors for linear grilles

Grill length (m)	Throw correction	Noise level correction (NR)
1 - 2	$L_T \times 1,00$	+ 0 NR
2 - 6,5	$L_T \times 1,10$	+ 5 NR

### SUPPLY:

#### Selection data:

Air flow rate  $q_v = 0,065$  m<sup>3</sup>/s  
Throw  $L_T = 4,5$  m at  $v_T = 0,30$  m/s

#### Solution:

Size H = 63 mm  
Supply air velocity  $v_k = 2,4$  m/s  
Noise level NR 21  
Total pressure with damper 100 % open:  $\Delta p_t = 10$  Pa  
Correction on noise level  
NR 21 + 10 = NR 31

### EXHAUST:

#### Selection data:

Air flow rate  $q_v = 0,1$  m<sup>3</sup>/s.m

#### Solution:

Size H = 75 mm  
Air velocity  $v_k = 4$  m/s  
Noise level NR 20  
Total pressure loss with damper 100 % open:  $\Delta p_t = 2,5$  Pa